

The radiated and discharge sound power levels of each unit at varying air flow rates and inlet static pressures are shown in the performance data tables. Disregarding other factors and/or equipment that could contribute to the noise in the occupied space, these ratings along with the acoustical environment in which the unit operates, will determine the perceived noise level.

Noise generated within the terminal and emitted through the discharge air (discharge sound) will be attenuated by any ductwork downstream of the terminal. The noise emitted through the casing of the terminal (radiated sound) will be attenuated by the room's ceiling. Depending upon the application, either the radiated or discharge noise level will be the relative higher and determine the perceived noise level in the occupied space. The occupied space itself will provide further attenuation depending on the acoustical characteristics of the walls, ceilings, floors and internal furnishings.

All manufacturers must make certain assumptions on the acoustical environment of the application and then apply these assumptions to the unit's sound power ratings to determine the resultant sound pressures and perceived noise level in the occupied space. While the AHRI sound power ratings have been certified and can accurately be compared from one manufacturer to another, the NC values predicted will be dependent upon the acoustical assumptions made.

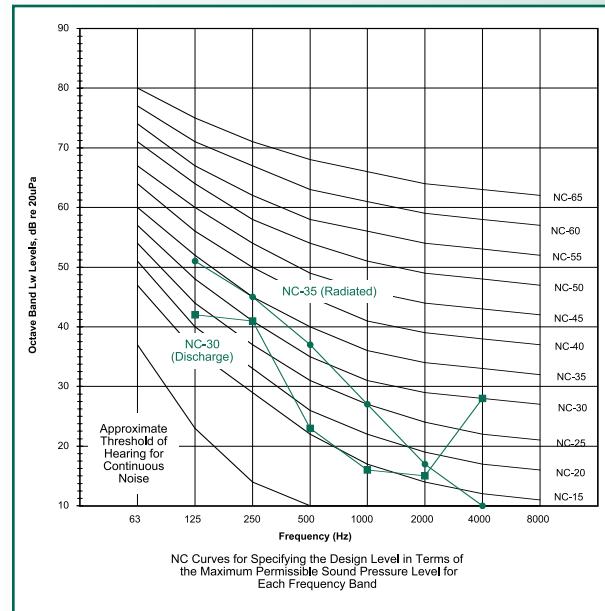
When selecting terminals, check the attenuation assumptions before comparing cataloged NC values. Anemostat uses the AHRI Standard 885, Appendix E attenuation assumptions for determining the anticipated noise levels. The attenuation assumptions in this standard are shown in table 2.

Table 3: Typical NC Design Values

Hotel rooms	25 - 35
Offices and conference rooms	25 - 35
Open offices	30 - 40
Classrooms	35 - 40 (max)
Churches	25 - 35
Hospital wards	30 - 40
Gymnasiums	40 - 45
Libraries	30 - 40

The NC curves are intended to reflect a human's perceived noise comfort. Plotting the anticipated sound pressure by octave band and determining the tangent NC curve reached throughout all octave bands (using the acoustical assumptions) will indicate the NC value anticipated.

Example of NC Curve Plot



Radiated Lw @ 1800 cfm - 1.0" w.g. Inlet Ps							
63	125	250	500	1000	2000	4000	8000
Lw Data	-----	69	64	57	53	48	46
Attenuation	-----	18	19	20	26	31	36
Plotted Data	-----	51	45	37	27	17	10
NC	-----	34	35	32	25	17	-----
Discharge 1800 cfm @ 0.25" w.g. External Ps							
63	125	250	500	1000	2000	4000	8000
Lw Data	-----	71	71	64	67	67	67
Attenuation	-----	29	30	41	51	52	39
Plotted Data	-----	42	41	23	16	15	28
NC	-----	22	30	16	-----	-----	30

Notes:

1. Size 7512 QST (see tables 39 and 40)
2. Radiated sound in the 250 Hz (third octave) is the controlling band

Table 2: AHRI Attenuation Table

Octave Band						
	2	3	4	5	6	7
Radiated	2	1	0	0	0	0
All Sizes	16	18	20	26	31	36
	18	19	20	26	31	36
Environmental Effect						
Type II Mineral Fiber						
Octave Band						
	2	3	4	5	6	7
Discharge	2	1	0	0	0	0
Sizes 5-7 (300-700 cfm)	2	4	10	20	20	14
	9	5	2	0	0	0
5 ft., Duct Lining (12x12)						
End Reflection						
5 ft., 8 in. Flex Duct						
5 ft., 8 in. Flex Duct						
Room Effect						
Sound Power Division						
Total dB Reduction						
Octave Band						
	2	3	4	5	6	7
Discharge	2	1	0	0	0	0
Sizes (>700 cfm)	2	3	9	18	17	12
	9	5	2	0	0	0
5 ft., Duct Lining (15x15)						
End Reflection						
5 ft., 8 in. Flex Duct						
Room Effect						
Sound Power Division						
Total dB Reduction						

